

Application of KULI for Study of Cooling System in Heavy Duty Truck

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Junghwan Lim



HYUNDAI·KIA MOTORS



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Background

□ Emission Control Plan in Korea

- EURO III in 2004
- EURO IV in 2006
- EURO V in 2008

➡ Modified Engine Configuration Requires Enhanced Cooling Performance

- Higher Heat Rejection By Coolant: 9.3% Increase
- Higher Pressure and Temperature for CAC Inlet:
Pr. 13.2%, Temp. 35.3% Increase

➡ Improve Cooling Package Capabilities

Model Set-Up

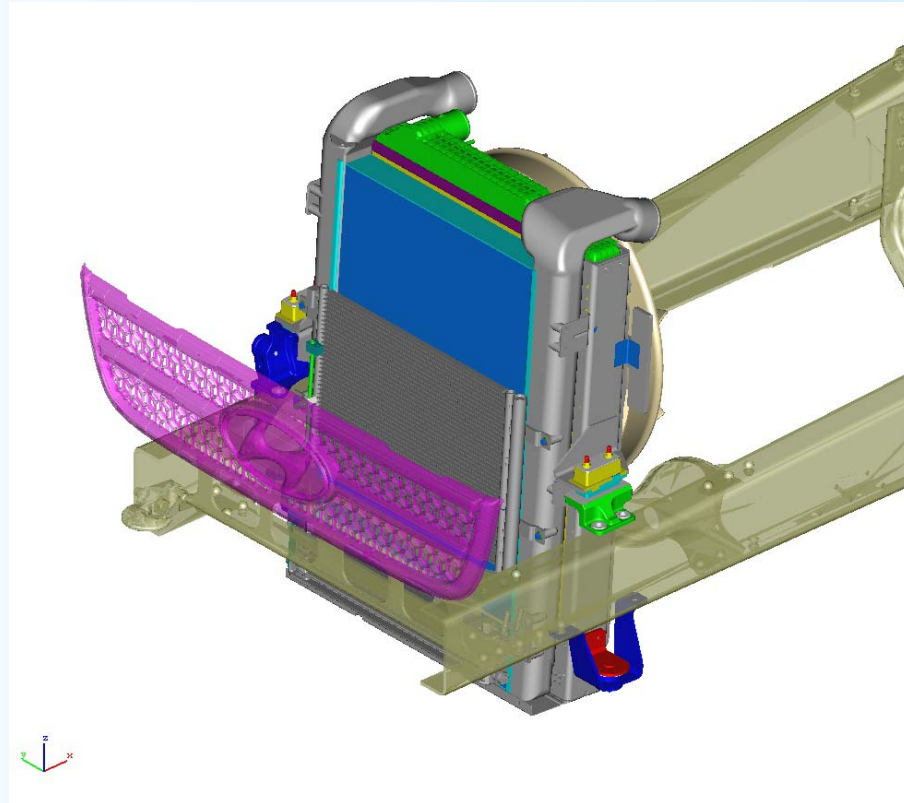
□ Engine Specifications

	Cooling Circuit			Charge Air Circuit		
	Mass Flow (kg/s)	Inlet Pr. (hPa)	Heat Rejection (kw)	Mass Flow (kg/s)	Inlet Pr. (hPa)	Inlet Temp. (°C)
Old Engine	7.91	2030	128.2	0.48	2680	157.4
04 EM Engine	7.49	2030	140.1	0.527	3033	213.0
+ /-(%)	+ 5.3	0	+ 9.3	+ 9.8	+ 13.2	+ 35.3

- Data for 04 EM Engine are estimated values

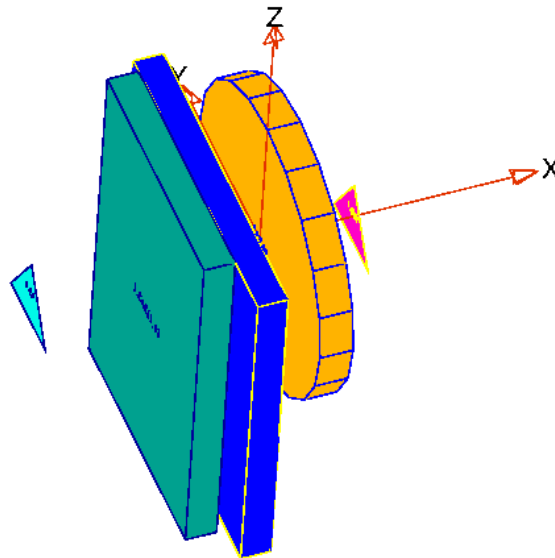
Model Set-Up

❑ Cooling Package Schematic



Model Set-Up

□ Air Side Lay-out In KULI



Model Set-Up

□ Major Characteristics of Model

- $C_p = 0.9$
- Radiator: Fin Tube type, -z direction
- CAC: Fin Tube type, -y direction
- BIR: Determined by correlation with Test data
- Fan: Eaton, Ring type, Transmission ratio,
Viscous Clutch Locked
- Driving Speed: 22 Km/h, 1800 RPM
- Air Conditioner is off
- Warm-up Temp.: 2°C

Model Set-Up

□ Model Correlation

- Determine Built-In-Resistance value to match ACT and IMTD values from the KULI Model to Test data, with an old engine



$$\Delta P \text{ (Pa)} = K Q^2 \text{ (m}^3\text{/s)}; \quad K = 90$$

	Radiator			Charge Air Cooler		
	T _{in} (°C)	T _{out}	ACT	T _{in}	T _{out}	IMTD
Test	91	86	39	163	50	25
Model	91.09	86.16	38.95	163.00	50.41	25.41

※ ACT: Air Clearance Temp = 105 – (Rad. Inlet Temp – Amb Temp.)

IMTD: Intake Manifold Temp. Difference = CAC Outlet Temp – Amb Temp

Analysis I

Cu Radiator

□ Target for ACT & IMTD

- ACT: 43°C ↑
- IMTD: 21°C ↓

□ 04EM Engine + Current Cooling Package

		Margin
ACT	42.17 °C	-0.83
IMTD	25.01 °C	-4.01
Cooling Air	3.093 kg/s	



Need to improve Cooling Performance of Radiator and CAC!!

Analysis I

□ Fan Drive Ratio Increase

1.14 → 1.33

		Margin
ACT	51.68 °C	+ 9.68
IMTD	21.85 °C	-0.85
Cooling Air	3.705 kg/s	



Target almost satisfied!
But IMTD need to be improved!

Analysis I

□ Radiator & CAC Size Increase

Approx. 10 % increase in Height

	Size (W×H×D, mm)	Core Type	Material
Rad.	729.5×897×66	Louver Fin (F.P = 2.8)	Cu
CAC	566×955×68	Wave Fin (F.P = 3.8)	Al

		Margin
ACT	41.59 °C	-1.41
IMTD	16.23 °C	+4.77
Cooling Air	3.049 kg/s	



ACT should be improved!

Analysis I

❑ Radiator & CAC Size Increase

Approx. 10 % increase in Height

❑ Rad. Fin Pitch Increase 2.80 → 3.68 mm

		Margin
ACT	42.29 °C	-0.71
IMTD	15.92 °C	+ 5.08
Cooling Air	3.142 kg/s	



ACT slightly improved, but still need to be improved!

Analysis I

❑ What we found out from Analysis I



Increasing sizes of the heat exchangers only helps improve IMTD, but not much for ACT.

To improve ACT, better increase cooling air flow.

❑ Radiator material change proposed



Due to weight saving and manufacturing issue in radiator supplier, Copper rad. is required to switch to Aluminum one.

Analysis II

Al Radiator

❑ Various Al Radiators Applied; Cu → Al

▪ Rad. Specs.

Case No.	Radiator Size (W× H× D, mm)	Core Type	Material
1	$718.2 \times 894 \times 48$	Wave Fin (F.P = 4.0)	Al
2	$718.2 \times 894 \times 60$	Wave Fin (F.P = 4.0)	Al
3	$718.2 \times 894 \times 48$	Louver Fin (F.P = 4.0)	Al
4	$718.2 \times 894 \times 60$	Louver Fin (F.P = 4.0)	Al

Analysis II

❑ Various AL Radiators Applied; Cu → Al

▪ Results

Case No.	ACT (°C)	IMTD (°C)	Air Flow (kg/s)
1	6.23 (-36.77)	17.50 (+ 3.5)	3.329
2	25.22 (-17.18)	17.63 (+ 3.37)	3.296
3	2.96 (-40.04)	17.46 (+ 3.54)	3.336
4	41.60 (-1.4)	18.36 (+ 2.64)	3.139

Analysis II

❑ Recommended Arrangement

AI Rad. Case 4 + Fan Drive Ratio 1.2

		Margin
ACT	44.89 °C	+ 1.89
IMTD	17.38 °C	+ 3.62
Cooling Air	3.357 kg/s	



Can meet the requirements !!

Conclusion

- ❑ The combination of AL Rad. ($718.2 \times 894 \times 60$, Louver Fin, F.P = 4.0), AL CAC ($566 \times 955 \times 68$, Wave Fin, F.P = 3.8), and Increased Fan Ratio 1.2 are recommended for the cooling package of a new engine.
- ❑ The characteristic data of the components are very crucial to accurate prediction of the cooling performance in KULI model.
- ❑ The results of KULI calculation should/will be compared with the test data to justify the use of BIR and air side simulation.